



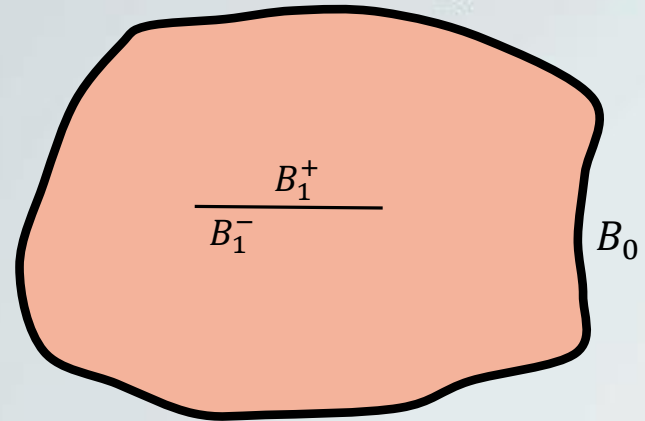
Conduction shape factor in exchanger tubes with a slit by using the meshfree boundary integral equation method

Hung-Wen Yang (楊洪聞), *Department of Civil Engineering, Tamkang University, Taiwan* (yhw10263@gmail.com)

Advisor: Prof. Jia-Wei Lee (李家瑋)

Abstract: Based on the successful experience of application the meshfree boundary integral equation method to calculating the conduction shape factor of heat exchanger tubes, this paper extends the meshfree boundary integral equation method to calculate the conduction shape factor of heat exchanger tubes containing a slit. The difference between the present method and the conventional boundary element method is that we use the local exact solution and Gaussian quadrature to technically determine the singular integral in the sense of Cauchy principal value and discretize the boundary integral equation, respectively. When using the present method to solve the problem containing a slit, we encounter the problem of degenerate boundary and twice occurrence of singular integral on the slit. To overcome above problems, we introduce the hypersingular integral equation to obtain independent equations and find a new local exact solution that can be suitable for the problem with a degenerate boundary. To avoid encountering the an certainty of jump function when using the Cartesian coordinate system to represent the local exact solution for a point on the slit, we adopt elliptical coordinates to construct the new local exact solution. After comparing the exact solutions of conduction shape factor provided in references [2] and [3], present results are consistent with those results. Finally, we also employ the conventional boundary element method (BEM) and finite element method (FEM) to calculate the temperature distribution on the cross-section of exchanger tubes. The present results make a good agreement with those results of BEM and FEM.

Problem description



Governing equation

Laplace equation : $\nabla^2 u(\mathbf{x}) = 0, \mathbf{x} \in D$

Outer and inner boundaries are isothermal condition

$$\begin{cases} u(x) = u_1 = 400k, & x \in B_1 \\ u(x) = u_0 = 300k, & x \in B_0 \end{cases}$$

\dot{Q} is the steady rate of heat transfer
 u_1 is the temperature of inner boundary
 u_0 is the temperature of outer boundary

Conduction Shape Factor: S

$$S = \frac{\dot{Q}}{L \times k \times (u_1 - u_0)}$$

L is the length of exchanger tube
 k is the thermal conductivity

Meshfree boundary integral equation method

$$0 = \int_B T(\mathbf{s}, \mathbf{x}) [u(\mathbf{s}) - w(\mathbf{s})] dB(\mathbf{s}) - \int_B U(\mathbf{s}, \mathbf{x}) [t(\mathbf{s}) - w_p(\mathbf{s})] dB(\mathbf{s}), \mathbf{x} \in B$$

$$0 = \int_B M(\mathbf{s}, \mathbf{x}) [u(\mathbf{s}) - w(\mathbf{s})] dB(\mathbf{s}) - \int_B L(\mathbf{s}, \mathbf{x}) [t(\mathbf{s}) - w_p(\mathbf{s})] dB(\mathbf{s}), \mathbf{x} \in B$$

$$t(\mathbf{s}) = \frac{\partial u(\mathbf{s})}{\partial n(\mathbf{s})} \quad w_p(\mathbf{s}) = \frac{\partial w(\mathbf{s})}{\partial n(\mathbf{s})} \quad \begin{matrix} B = B_0 \cup B_1 \\ B_1 = B_1^+ \cup B_1^- \end{matrix}$$

For $\mathbf{x} \in B_0 \Rightarrow$ Ordinary boundary

$$0 = \int_B T(\mathbf{s}, \mathbf{x}) [u(\mathbf{s}) - w(\mathbf{s})] dB(\mathbf{s}) - \int_B U(\mathbf{s}, \mathbf{x}) [t(\mathbf{s}) - w_p(\mathbf{s})] dB(\mathbf{s}), \mathbf{x} \in B_0$$

$$w(\mathbf{s}) = u(\mathbf{x}) + [(s_x - x)n_x(\mathbf{x}) + (s_y - y)n_y(\mathbf{x})]t(\mathbf{x}).$$

Gaussian integral

$$0 = \sum_{j=1}^{N_g} \omega_j T(\mathbf{s}_j, \mathbf{x}_i) [u_j - w_j^i] j(\xi_j) - \sum_{j=1}^{N_g} \omega_j U(\mathbf{s}_j, \mathbf{x}_i) [t_j - w_p^i] j_b(\xi_j)$$

$$j_b(\xi) = \left(\frac{\tau_L - \tau_0}{2} \right) \sqrt{(x'(\tau))^2 + (y'(\tau))^2} \Big|_{\tau=\tau(\xi)}$$

For $\mathbf{x} \in B_1 \Rightarrow$ Degenerate boundary

$$\begin{aligned} & \mathbf{x}^+ \in B_1^+ \\ & 0 = \int_{B_1} T(\mathbf{s}, \mathbf{x}) [u(\mathbf{s}) - w(\mathbf{s})] dB(\mathbf{s}) - \int_{B_1} U(\mathbf{s}, \mathbf{x}) [t(\mathbf{s}) - w_p(\mathbf{s})] dB(\mathbf{s}) \\ & + \int_{B_0} T(\mathbf{s}, \mathbf{x}) u(\mathbf{s}) dB(\mathbf{s}) - \int_{B_0} U(\mathbf{s}, \mathbf{x}) t(\mathbf{s}) dB(\mathbf{s}), \mathbf{x} \in B_1^+ \end{aligned}$$

$$w(\mathbf{s}) = f_1(\mathbf{s}, \mathbf{x}^+) u(\mathbf{x}^+) + f_2(\mathbf{s}, \mathbf{x}^+) u(\mathbf{x}^-) + g_1(\mathbf{s}, \mathbf{x}^+) t(\mathbf{x}^+) + g_2(\mathbf{s}, \mathbf{x}^+) t(\mathbf{x}^-)$$

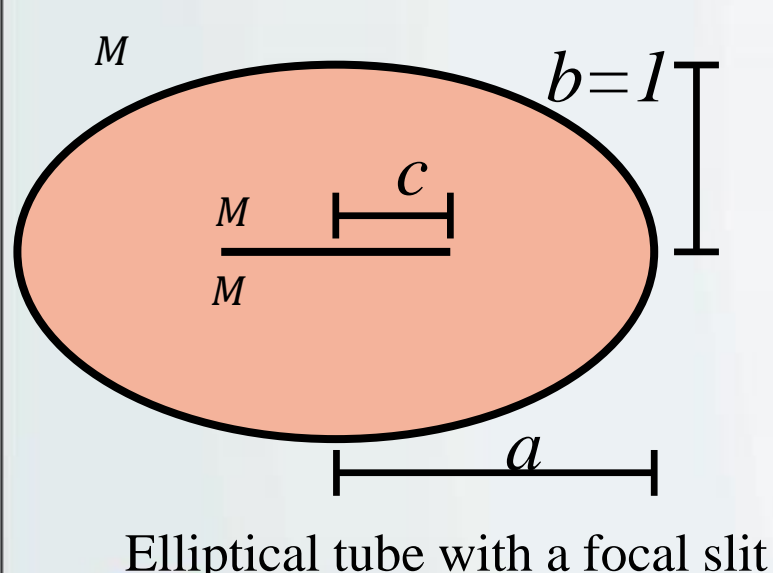
$$\begin{aligned} & \mathbf{x}^- \in B_1^- \\ & 0 = \int_{B_1} M(\mathbf{s}, \mathbf{x}) [u(\mathbf{s}) - w(\mathbf{s})] dB(\mathbf{s}) - \int_{B_1} L(\mathbf{s}, \mathbf{x}) [t(\mathbf{s}) - w_p(\mathbf{s})] dB(\mathbf{s}) \\ & + \int_{B_0} M(\mathbf{s}, \mathbf{x}) u(\mathbf{s}) dB(\mathbf{s}) - \int_{B_0} L(\mathbf{s}, \mathbf{x}) t(\mathbf{s}) dB(\mathbf{s}), \mathbf{x} \in B_1^- \end{aligned}$$

$$w(\mathbf{s}) = f_1(\mathbf{s}, \mathbf{x}^-) u(\mathbf{x}^+) + f_2(\mathbf{s}, \mathbf{x}^-) u(\mathbf{x}^-) + g_1(\mathbf{s}, \mathbf{x}^-) t(\mathbf{x}^+) + g_2(\mathbf{s}, \mathbf{x}^-) t(\mathbf{x}^-)$$

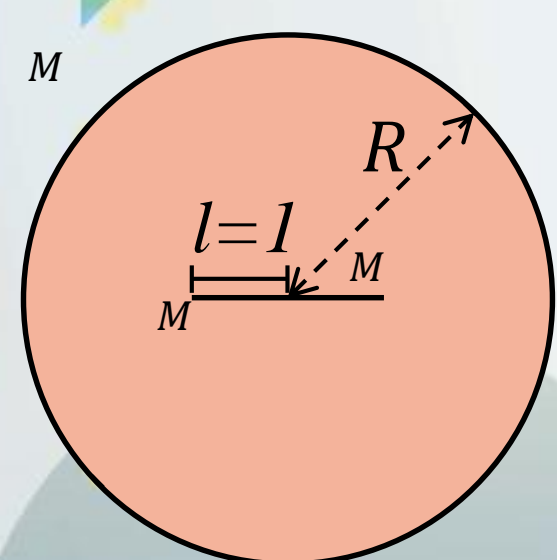
Results and discussion

Shape function of local exact solution for the problem with a degenerate boundary

$f_1(\mathbf{s}, \mathbf{x}^+) = \frac{e^{\xi s}}{\cos(\eta_{x^+})} e^{-\xi s} \cos(\eta_s) - \frac{1}{2} \frac{e^{2\xi s}}{\cos(2\eta_{x^+})} e^{-2\xi s} \cos(2\eta_s) + \frac{e^{\xi s}}{\sin(\eta_{x^+})} e^{-\xi s} \sin(\eta_s) - \frac{1}{2} \frac{e^{2\xi s}}{\sin(2\eta_{x^+})} e^{-2\xi s} \sin(2\eta_s)$	$g_1(\mathbf{s}, \mathbf{x}^+) = -\frac{1}{2} J(\mathbf{x}^+) \left[\frac{e^{\xi s}}{\cos(\eta_{x^+})} e^{-\xi s} \cos(\eta_s) - \frac{e^{2\xi s}}{\cos(2\eta_{x^+})} e^{-2\xi s} \cos(2\eta_s) + \frac{e^{\xi s}}{\sin(\eta_{x^+})} e^{-\xi s} \sin(\eta_s) - \frac{e^{2\xi s}}{\sin(2\eta_{x^+})} e^{-2\xi s} \sin(2\eta_s) \right]$	$f_1(\mathbf{s}, \mathbf{x}^-) = \frac{e^{\xi s}}{\cos(\eta_{x^-})} e^{-\xi s} \cos(\eta_s) - \frac{1}{2} \frac{e^{2\xi s}}{\cos(2\eta_{x^-})} e^{-2\xi s} \cos(2\eta_s) + \frac{e^{\xi s}}{\sin(-\eta_{x^-})} e^{-\xi s} \sin(\eta_s) - \frac{1}{2} \frac{e^{2\xi s}}{\sin(-2\eta_{x^-})} e^{-2\xi s} \sin(2\eta_s)$	$g_1(\mathbf{s}, \mathbf{x}^-) = -\frac{1}{2} J(\mathbf{x}^-) \left[\frac{e^{\xi s}}{\cos(\eta_{x^-})} e^{-\xi s} \cos(\eta_s) - \frac{e^{2\xi s}}{\cos(2\eta_{x^-})} e^{-2\xi s} \cos(2\eta_s) + \frac{e^{\xi s}}{\sin(-\eta_{x^-})} e^{-\xi s} \sin(\eta_s) - \frac{e^{2\xi s}}{\sin(-2\eta_{x^-})} e^{-2\xi s} \sin(2\eta_s) \right]$
$f_2(\mathbf{s}, \mathbf{x}^+) = \frac{e^{\xi s}}{\cos(\eta_{x^+})} e^{-\xi s} \cos(\eta_s) - \frac{1}{2} \frac{e^{2\xi s}}{\cos(2\eta_{x^+})} e^{-2\xi s} \cos(2\eta_s) - \frac{e^{\xi s}}{\sin(\eta_{x^+})} e^{-\xi s} \sin(\eta_s) + \frac{1}{2} \frac{e^{2\xi s}}{\sin(2\eta_{x^+})} e^{-2\xi s} \sin(2\eta_s)$	$g_2(\mathbf{s}, \mathbf{x}^+) = -\frac{1}{2} J(\mathbf{x}^+) \left[\frac{e^{\xi s}}{\cos(\eta_{x^+})} e^{-\xi s} \cos(\eta_s) - \frac{e^{2\xi s}}{\cos(2\eta_{x^+})} e^{-2\xi s} \cos(2\eta_s) - \frac{e^{\xi s}}{\sin(\eta_{x^+})} e^{-\xi s} \sin(\eta_s) + \frac{e^{2\xi s}}{\sin(2\eta_{x^+})} e^{-2\xi s} \sin(2\eta_s) \right]$	$f_2(\mathbf{s}, \mathbf{x}^-) = \frac{e^{\xi s}}{\cos(\eta_{x^-})} e^{-\xi s} \cos(\eta_s) - \frac{1}{2} \frac{e^{2\xi s}}{\cos(2\eta_{x^-})} e^{-2\xi s} \cos(2\eta_s) - \frac{e^{\xi s}}{\sin(-\eta_{x^-})} e^{-\xi s} \sin(\eta_s) + \frac{1}{2} \frac{e^{2\xi s}}{\sin(-2\eta_{x^-})} e^{-2\xi s} \sin(2\eta_s)$	$g_2(\mathbf{s}, \mathbf{x}^-) = -\frac{1}{2} J(\mathbf{x}^-) \left[\frac{e^{\xi s}}{\cos(\eta_{x^-})} e^{-\xi s} \cos(\eta_s) - \frac{e^{2\xi s}}{\cos(2\eta_{x^-})} e^{-2\xi s} \cos(2\eta_s) - \frac{e^{\xi s}}{\sin(-\eta_{x^-})} e^{-\xi s} \sin(\eta_s) + \frac{e^{2\xi s}}{\sin(-2\eta_{x^-})} e^{-2\xi s} \sin(2\eta_s) \right]$

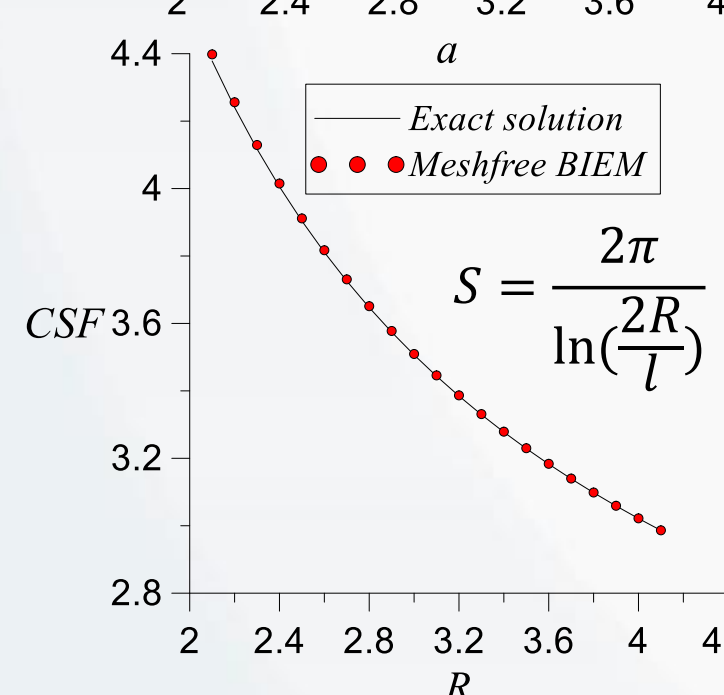
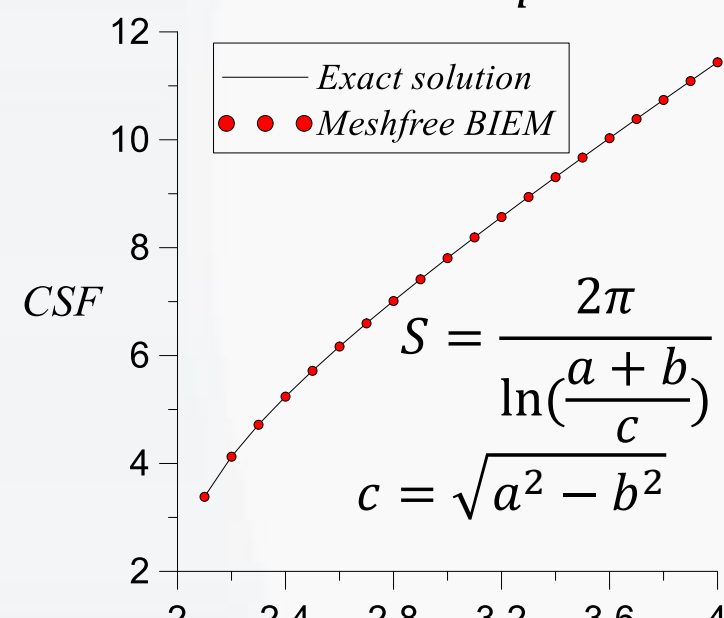


Elliptical tube with a focal slit

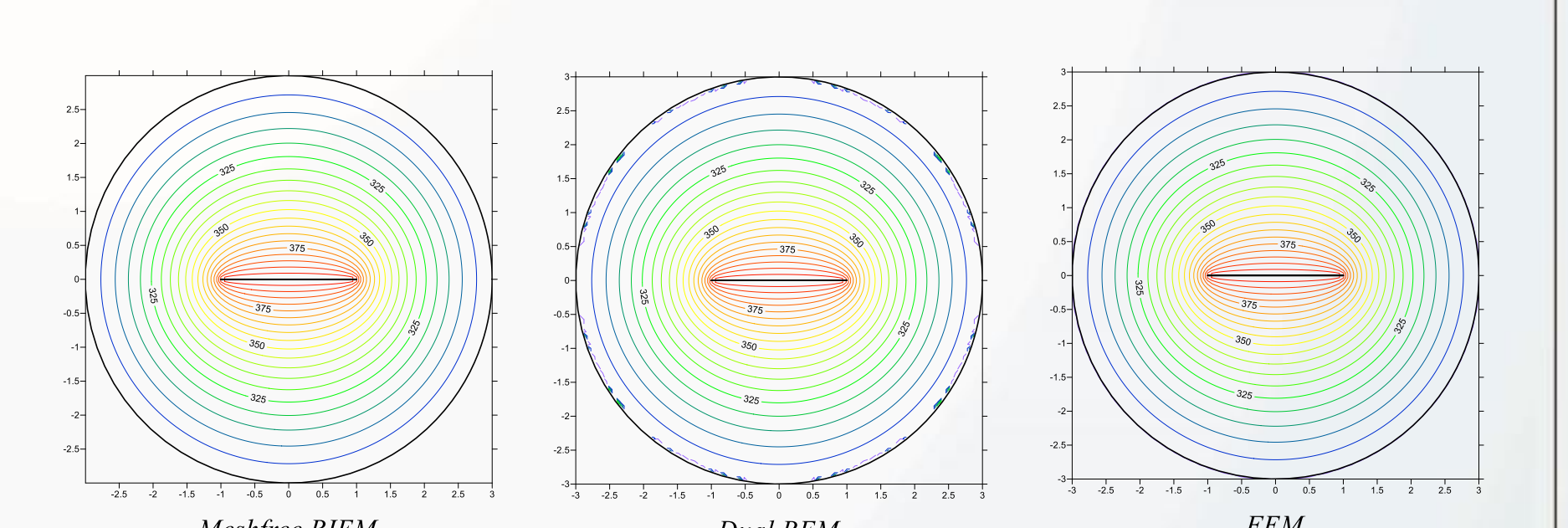
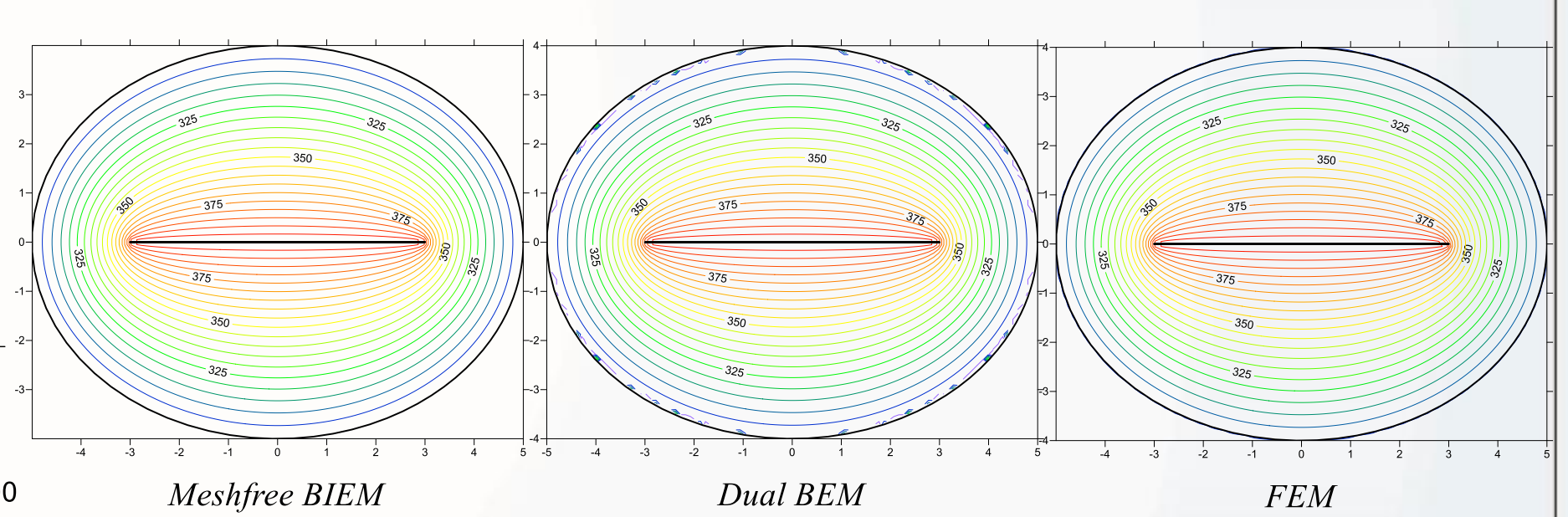
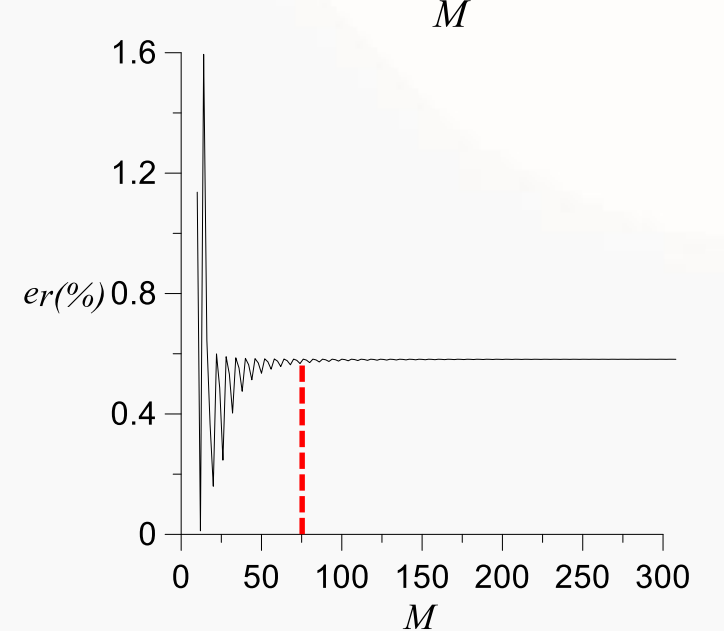
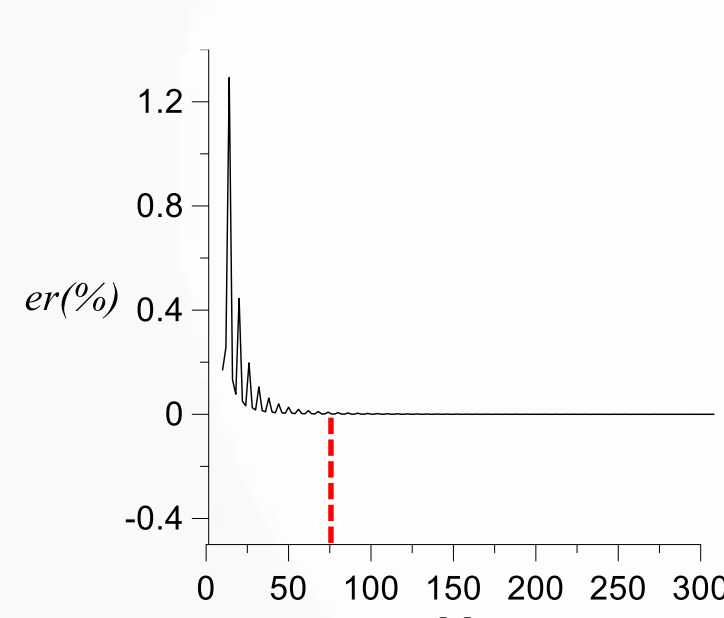


Circular tube with a central slit

Conduction Shape Factor



Relative error



Conclusions

1. The conduction shape factor of exchanger tubes with a slit can be successfully calculated by using the meshfree boundary integral equation method.
2. To overcome the issue of degenerate boundaries, it is necessary to use the hypersingular integral equation.
3. The appropriate local exact solution for a problem with a degenerate boundary can be constructed by using the elliptical coordinates.

References

- [1] Lee J. W., Yang H. W. and Chen J. T., 2023, "Steady state heat conduction in exchanger tubes by using the meshfree boundary integral equation method: conduction shape factor and degenerate scale," J of Heat Transfer-Transactions of the ASME, accepted.
- [2] Balcerzak M. J. and Raynor S., 1961, "Steady state temperature distribution and heat flow in prismatic bars with isothermal boundary conditions", Int. J. Heat Mass Transfer. Vol. 3, pp.113-125.
- [3] Lewis G. K., 1968, "Shape factors in conduction heat flow for circular bars and slabs with various internal geometries", Int. J. Heat Mass Transfer. Vol. 11, pp.985-992.
- [4] Hong H. K. and Chen J. T., 1988, Derivations of Integral Equations of Elasticity, Journal of Engineering Mechanics, ASCE, Vol.114, No.6, pp.1028-1044.
- [5] Chen J. T. and Hong H. K., 1999, Review of dual boundary element methods with emphasis on hypersingular integrals and divergent series, Applied Mechanics Reviews, ASME, Vol.52, No.1, pp.17-33.

